



## HETEROGENEOUS INTEGRATION ROADMAP

**2021 Edition**

# Chapter 20: Thermal

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# Chapter 20: Thermal

## 1.0 Introduction and Scope

Heterogeneous Integration poses several significant challenges for thermal management at multiple length scales ranging from heat extraction from hot spots, heat transfer through multiple layers of materials, different target temperatures for specific devices/materials, to heat rejection to a system cooling solution or the ambient. This Technology Working Group (TWG) will consider three areas for thermal management:

- Die level;
- Package integration/System-in-Package (SIP)/module level;
- System level (limited to board and server level).

In addition to the taxonomy of the physical categories listed above, this chapter will focus on articulating the following in quantitative (wherever possible) and qualitative terms:

- Canonical problems with thermal challenges;
- Cooling limits for known solutions;
- Advanced concepts and research.

## 2.0 Canonical Problems with Thermal Challenges

The canonical thermal problems and challenges presented in detail herein are:

- 2D chip with stacked memory on a silicon/glass interposer.
- 3D stacked die with conduction interfaces (*inclusion of 3D stacked die High Bandwidth Memory (HBM) devices as an example*).
- 3D stacked die with embedded liquid cooling.
- Optics/photonics-based heterogeneous package.
- Harsh environment (military, aerospace, automobile).
- Mobile application chipset (package-on-package, fan-out, bridge).
- Voltage regulators in a heterogeneous package.

Prior to discussing the Canonical problems in detail per the discussion from the 2019 version of this chapter, and to provide an illustration of the dramatic future thermal challenges in the area of thermals, *nominal* Thermal Demand Envelopes are displayed in Figure 1 [1], which indicate that Thermal Technologies must cover a Hotspot Density Envelope in the 2x-4x range (higher than the supported average power density), with an understanding of the impact of hotspot area relative to total die area, overall thermal design power and upside capability for both 2D and 3D Architectures.

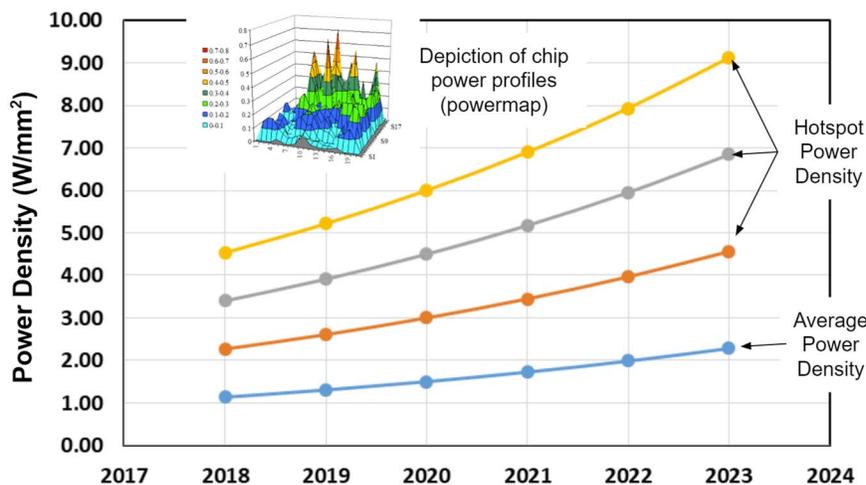


Figure 1: Nominal Thermal Demand Envelopes covering average and hotspot peak power density for both 2D and 3D Architectures [1 W/mm<sup>2</sup> = 100 W/cm<sup>2</sup>]

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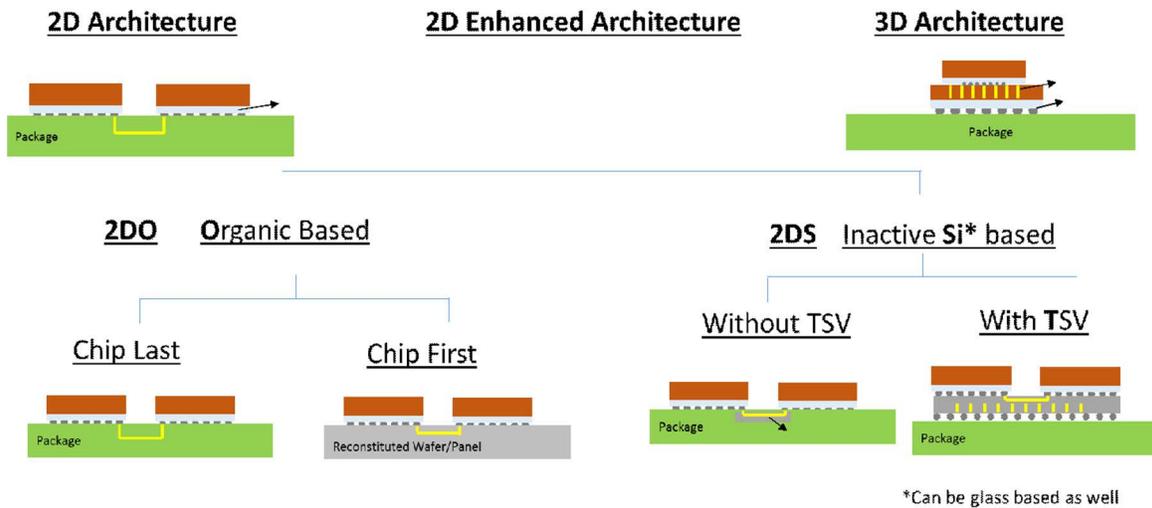
It is important for the thermal community to identify and develop a detailed understanding of the capabilities and limitations of key thermal technologies that meet or exceed these demands so that they are available well in advance of need and can be implemented if they meet integration cost envelopes.

**2.1 Thermal challenges in 2D Packages**

Figure 2 shows the 2D enhanced architecture (2D-EA) as a part of the overall 2D-3D packaging taxonomy. 2D-EA can be divided between 2D Organic (2DO) and 2D Passive Silicon (2DS).

The 2DO architecture can be further divided to Chip Last and Chip First. Chip Last is also known as RDL (redistribution layer) first and is a process where the device wafer is bumped and then diced, and flip-chipped to RDL that is formed on a temporary carrier. Chip First is a process where dice are reconstructed on a wafer, embedded in mold compound and followed by RDL forming process. For detailed discussion, see Chapter 23 on WLP.

The 2DS architecture can further be divided between without and with through-silicon vias (TSVs).



\*Can be glass based as well

Figure 2: 2D-3D packaging taxonomy [2].

2D-EA is a side-by-side heterogeneous integration of two or more functional components (ASIC, FPGA, CPU, GPU, single or 3D-stacked memory) using an organic or inorganic interposer or an embedded high-density interconnect-enabling connector (e.g. Embedded Multi-die Interconnect Bridge (EMIB), as shown in Figure 3.

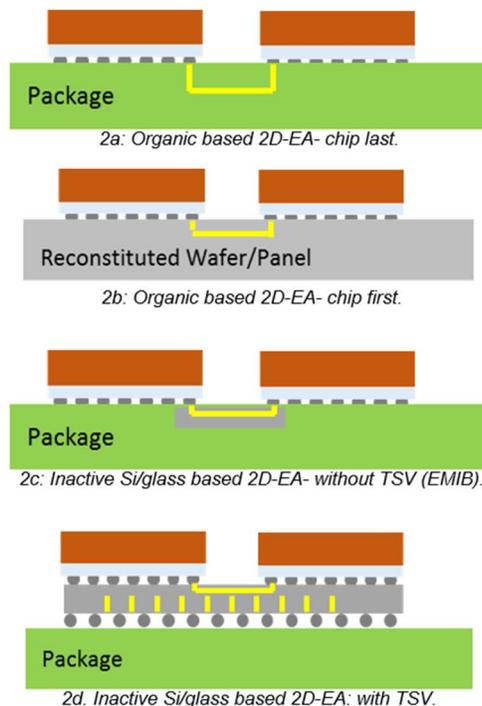


Figure 3: Schematic of 2D-EA [2].

In summary, the thermal challenges for the 2D heterogeneous-integrated packages are listed below:

- Increasing package power density;
- Increasing total package power dissipation;
- Thermal cross-talk, including the need for thermal isolation;
- Different thermal ( $T_j$ ) requirements and sensitivities;
- Thermal interface material (TIM1 or TIM1.5) thermal insulance ( $K\text{-mm}^2/\text{W}$ ) uncertainty from increasing form factor and Si surface flatness and overall warpage impact;
- Enabling good thermo-mechanical performance;
- Interposer thermal properties (glass/Si/organic) (see Figure 3) including anisotropy;
- Interposer thermal conductivity has a strong impact on the chip thermal resistance;
- Glass and Si interposer performance can be made comparable, by appropriate enhancements;
- Interposer heat spreading and heat removal.

Figure 4 depicts several different cases that are analyzed, with the thermal results mapped in Figure 5.

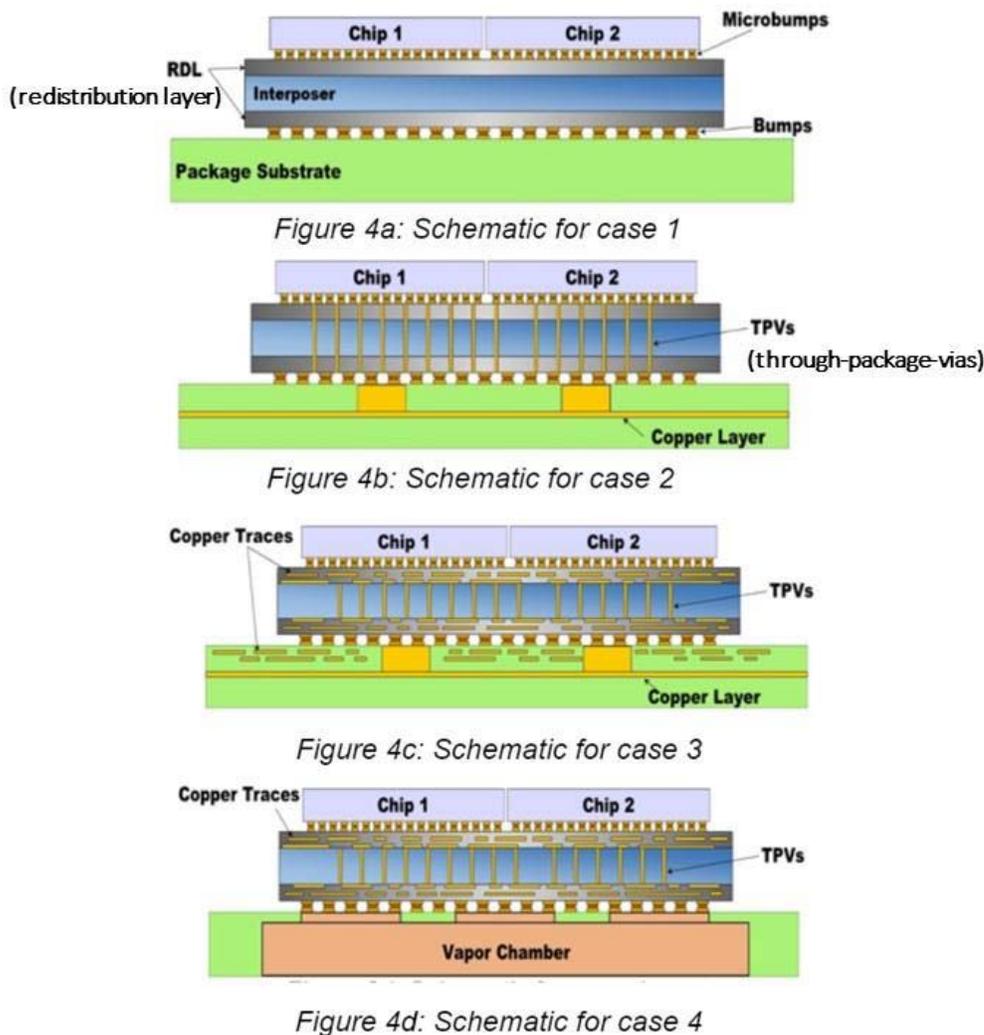


Figure 4: Multiple 2D configurations for thermal assessment [2].

Figure 5 displays the results from analyzing cases 4a - 4d depicted in Figure 4 and illustrates that with internal thermal enhancements, and coupling with effective heat removal pathways, glass interposer performance could be made comparable to silicon interposers. Glass interposers have other attractive features including better high frequency signaling characteristics, and lower manufacturing costs due to panel fabrication. So this thermal enhancement may make glass a more attractive technology for the reduction in junction-to-case thermal resistance ( $K/W$ ).

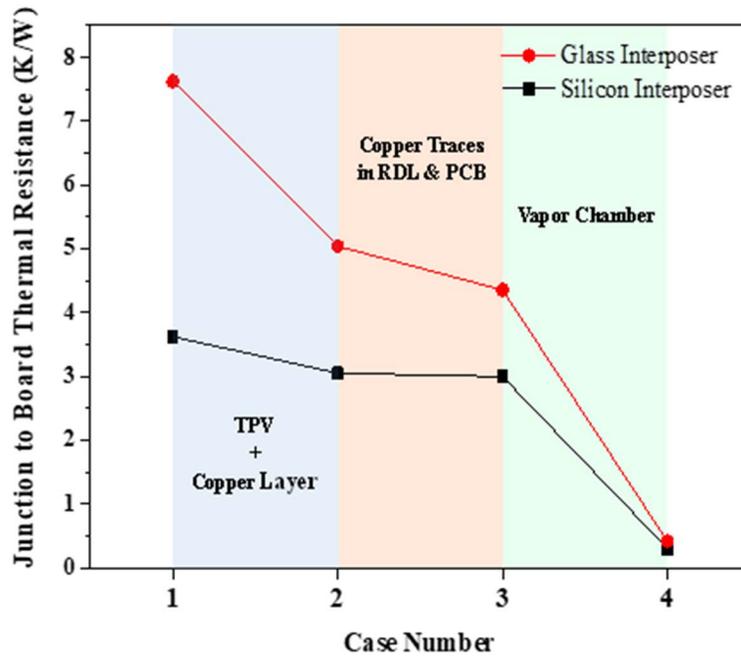


Figure 5: Impact of interposer and substrate thermal conductivity on package thermal resistance [3].

**2.2 3D Stacked-Chip Packages with Conduction Interfaces**

Thermal management is critical in the design of high-power 3D stacked-chip packages which enable high-bandwidth and low-latency communications. By stacking chips, as shown in Figure 6, the effective power density increases because the power generated in the 3D stack has to be dissipated over the “footprint” area of a single chip.

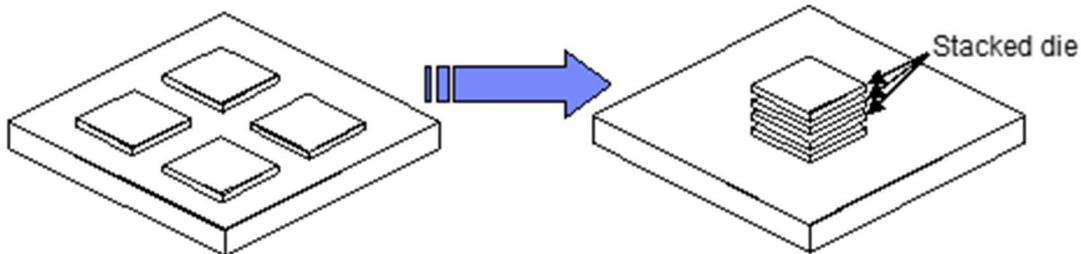


Figure 6. Effective power density increases in a 3D stacked package

In a high-power 2D package, shown in Figure 7, most of the heat generated in the chip conducts through a thermal interface material into a metal lid and then externally to a heat sink.

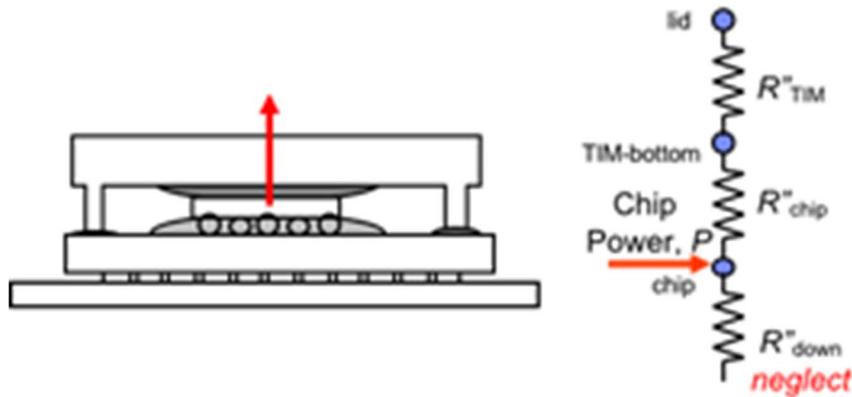


Figure 7. Conduction paths in a high-power 2D package

In a high-power 3D package, heat from the bottom chips in the 3D stack has to conduct through additional materials that form the interconnect/underfill layers, the back-end-of-the-line (BEOL) layers, and the bulk silicon above the BEOL layers. This causes additional thermal resistances in the heat conduction path as depicted in Figure 8.

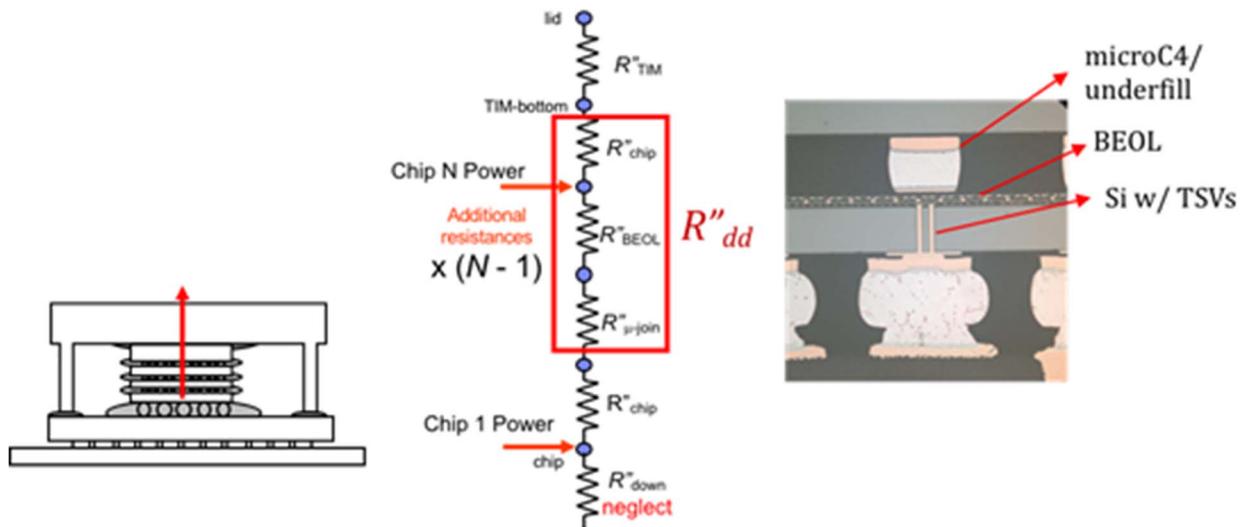


Figure 8. Conduction paths in a high-power 3D package

Additional increase in temperature in a 3D chip package can occur due to vertical alignment of high-density computing cores and different-sized stacked chips. Alternatively, such a chip stack can be cooled from both top and bottom, offering a 2x improvement in cooling capacity but at the cost of a far more complex design.

A highly relevant example of a 3D stacked die package that is prevalent in the industry is the High Bandwidth Memory (HBM) that is depicted in Figure 9 [4]. Figure 9 illustrates a sample System-in-Package that comprises a CPU or GPU Processor and a stacked die HBM, on a shared Silicon Interposer, with the sub-assembly residing on a PCB. As may be surmised from the preceding discussion of Figure 8, the stacked-die HBM poses significant cooling challenges for the removal of the heat dissipated through conduction in the vicinity of the heat sources. As is also evident from Figure 9, the efficacy of heat removal from such stacked dies can enable increased memory capacity through greater stacking of memory dies (2- to 4- to 8-High stacks).

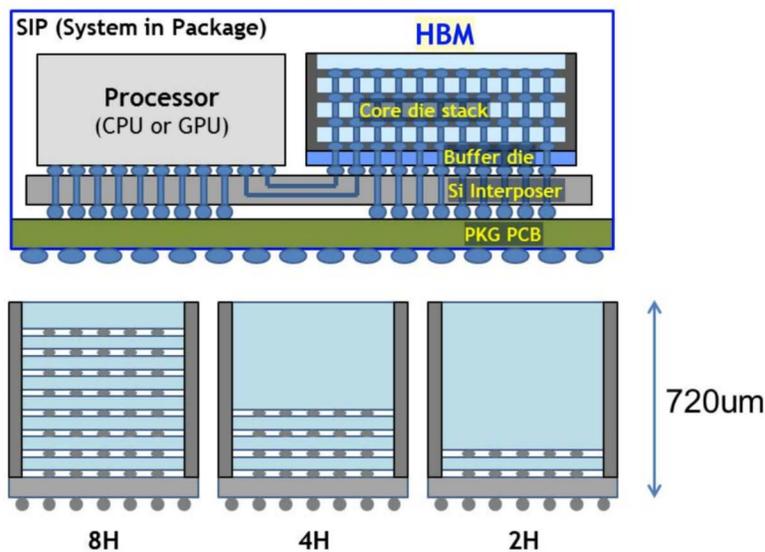


Figure 9. Example of a Heterogeneously Integrated Package with 3D stacked die High Bandwidth Memory. Examples of 2, 4, and 8 High Stacked Die HBMs [4]

### 2.3 3D Stacked Die with Embedded Liquid Cooling

To continue scaling computer performance as Moore’s Law transistor scaling slows, the IT industry has turned to 3D chip stack interconnect technology, which both increases the bandwidth between chips and enables heterogeneous integration to improve computing performance. However, the stacking of chips presents new thermal challenges as heat generated by multiple chips within a chip stack results in higher heat density which must be effectively removed.

The traditional approach of 2D chip thermal management is to conduct the heat from the active devices through the silicon die to a heatsink or cold plate which is attached to the top (or backside) of the chip with a thermal interface material as shown in Figure 10(a). The conduction of heat through a 3D chip stack as shown in Figure 10(b) creates

a thermal challenge, since the heat must conduct through multiple dies within the stack. Heat generated by the  $N$ th die in a chip stack will conduct heat through  $N-1$  dies to reach the heatsink or cold plate placed upon the top of the first die in the chip stack. It is important to note that the dies include complex structures including the Front-End-of-the-Line (FEOL), Back-End-of-the-Line (BEOL) with multiple wiring levels, and Through-Silicon-Vias with  $\mu$ C4 interconnects between them, as also shown in Figure 10b. These structures, when stacked and assembled with the usual placement and alignment tolerances, dramatically increase the thermal resistance between the dies in the stack and the heatsink or cold plate placed on the top die in the stack.

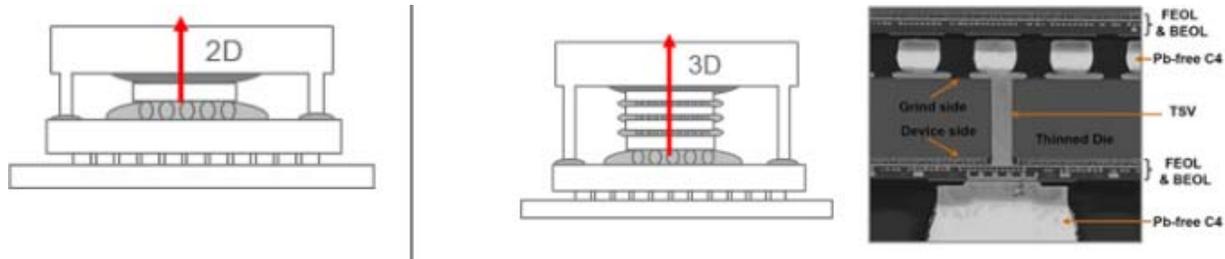


Figure 10: 2D vs 3D Thermal Path

While conducting heat through a chip stack can provide effective thermal management for stacks of low-power chips, when stacks include one or more high-power chips within the stack the conduction of heat through the dies and interconnects in the stack can produce high gradients in the chip junction temperatures across the stack. A method to address the challenges of cooling 3D chip stack structures described in this section is embedded cooling, where coolant flows either within (intrachip) or between (interchip) the stacked high-power chips as shown in Figure 11.

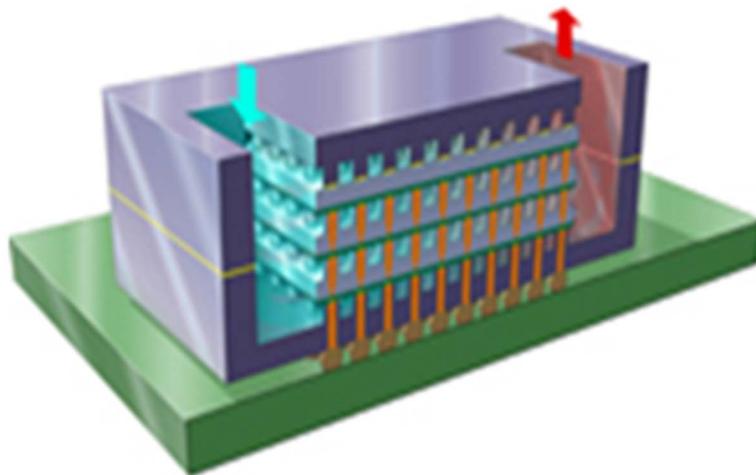


Figure 11 Embedded cooling with liquid flow internal to a 3D stacked chip package

Embedded cooling creates new requirements for 3D chip stack co-design, including placement and dimensions of fluidic channels, coolant properties, and compatibility with chip stack power and signal electrical interconnects. The fluid channel heights may be guided by compatibility of fabrication processes to create electrical interconnects between dies in the stack layers. For example, parallel channels with dimensions which meet the electrical requirements can create substantial pressure drop when using a single-phase liquid to flow across 20 mm (or more) for large processor dies. Use of the most conventional liquid coolant, water (which is conductive), requires the need to isolate the fluid from power and signal electrical interconnects and also consider dielectric losses associated with transmitting high frequency signals nearby. These challenges are part of the electrical, mechanical and thermal co-design of embedded cooling into a 3D chip stack.

#### 2.4 Thermal challenges in Photonic devices

Photonic devices as shown in Figure 12 can have high power density and require novel approaches to thermal solutions. Thermal challenges associated with photonic devices pertain to both the temperature swing and the absolute operating temperature of a given device. Depending on the component and use conditions, either one or both criteria may limit its performance. For example, optical modulators often depend on interference and resonant effects. Resonant modulators, which are useful to reduce the energy per bit, are mainly limited by the temperature

swing (typically  $< 30^{\circ}\text{C}$ ), whereas oscillators such as laser sources are limited by both the temperature swing (typically  $< 30^{\circ}\text{C}$ ) and the absolute operating temperature (typically  $< 100^{\circ}\text{C}$ ) [5]. In addition, photonic devices can be integrated with other functional IC components at the chip and/or package/system level. These components may have different thermal specifications which require both package and system thermal solution optimization.

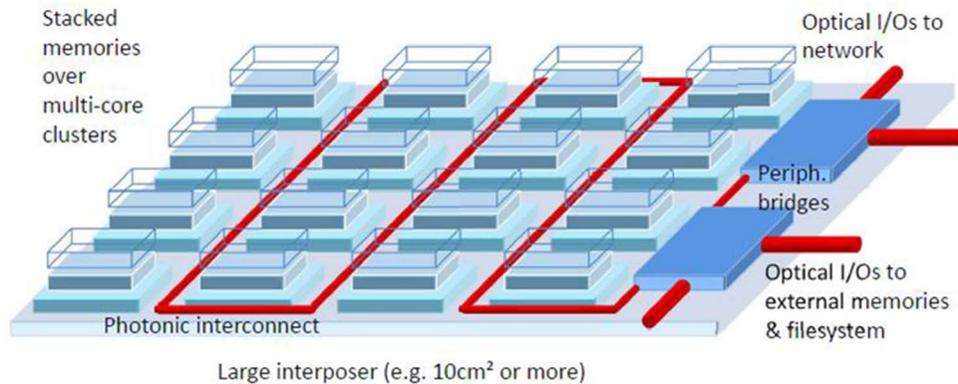


Figure 12: Heterogenous packaged photonics with stringent temperature control challenges [6]

## 2.5 Heterogeneous Integration for Harsh Environments

### 2.5.1. Aerospace, Automotive and Space Harsh Environment Requirements

As the world is getting more digital – through such trends as IoT, autonomous vehicles, AI and electric drive – safe and reliable operation of electronics in harsh environments such as aerospace, automotive, and space are becoming more and more critical.

In order to let compute systems take control of assets that can impact the safety of persons and/or cost millions of dollars, strict standards of hardware and software system safety must be taken into account. Harsh environment electronic applications must typically satisfy stringent requirements to operate safety-critical applications in extreme temperature, dust, vibration, and corrosive environments for operational periods of 10 to 30 years. In an example of harsh environment electronics, aerospace class I electronics need to operate in a continuous ambient environment of  $-54^{\circ}\text{C}$  to  $55^{\circ}\text{C}$  and survive a temporary 30-minute ambient environment of  $71^{\circ}\text{C}$ , without cooling air supply, simulating a cooling air supply fault condition.

In the automotive environment, burn-in and test of heterogeneous or multichip systems is a particular challenge. Test temperatures are extreme because of the harsh environments that these systems must operate in, sometimes ranging from  $-55^{\circ}\text{C}$  to  $175^{\circ}\text{C}$ . For automotive power electronics, the research focus and target is to increase the power density to 100 kW/L and double the reliability from 150,000 to 300,000 miles, while lowering the cost of power electronics, electric motors and the overall traction drive system by 2025 [7]. This power electronics power density increase to 100 kW/L represents a factor of 5 to 10 increase with respect to the state-of-the-art electric-drive vehicle power electronics.

Satellites are increasingly using power-hungry devices such as earth observation detectors operating in new or multiple frequency bands. The higher frequency bands currently under consideration for next-generation transmission systems offer higher bandwidths than the longer-wavelength bands currently in use. At the same time, those shorter-wavelength bands by nature have larger propagation losses. Challenges in space requirements involve the impact of solar heating and space radiation with the potential of significant thermal cycling of the electronics package, which is especially challenging for heterogeneously integrated packages.

Application of heterogeneous integration to harsh environments is challenging due to the 3D top-side contour of heterogeneous packages, the dissimilar junction limits of heterogeneous components in a system, and the extreme thermomechanical challenges imposed by the extreme temperature cycles. However, due to the superior performance, it is likely only a matter of time before these challenges are overcome and heterogeneously integrated chips make their way into the harsh environment space.

### 2.5.2 Heterogeneous Integration Top Side Cooling Solution needs

Rugged harsh-environment electronics rely predominantly for heat rejection through a thermal interface layer and a heat spreader mounted over the flip-chip package. This thermal interface layer serves to thermally connect components of dissimilar coefficient of thermal expansion (CTE) such as silicon (chip) and aluminum or copper (heat spreader). As ruggedized electronics typically are produced in moderate volumes (100s-1000s), chip height, chip

warp and other non-planarity variations batch to batch of several hundreds of microns can be typical due to ball grid array (BGA) and silicon variations. The thermal interface material also serves to compliantly compensate for these differences by filling gaps and ensuring good thermal contact between these components.

As heterogeneous integration introduces 3D non-planar and large silicon structures, challenges can be envisioned in connecting to a top-side heat sink using current thermal interface materials. It can also be expected that as multiple micro-BGA connections are used to create vertical stacks, height and planarity variations will accumulate, resulting in amplified variance in the eventual location of the top-side chip interface as illustrated in Figure 13.

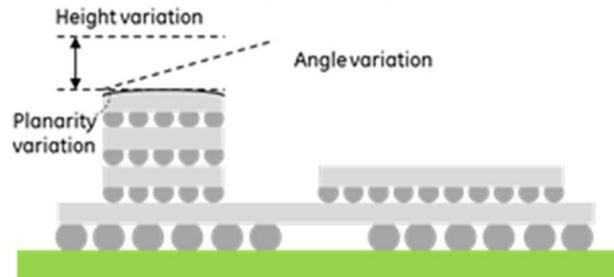


Figure 13: Notional 3D chip architecture and anticipated topology challenges

To manage this, novel 3D thermal interface material systems are envisioned. De Bock et al. [8] describes such a thermal interface system that uses a thicker layer of low melting point (LMP) solder, encapsulated by an ultra-thin micro layer of a high-temperature polymer. When pressure is applied during heating, the LMP solder flows and conforms to the chip shape while being contained by the thin polymer “bag” as illustrated in Figure 14. When the containment layer polymer is sufficiently thin, its detrimental contribution to the TIM thermal resistance can be sufficiently small, outweighed by the superior thermal conductivity of LMP solders like Indium ( $k \sim 70 \text{ W/m-K}$ ), which exceeds common thermal interface materials.

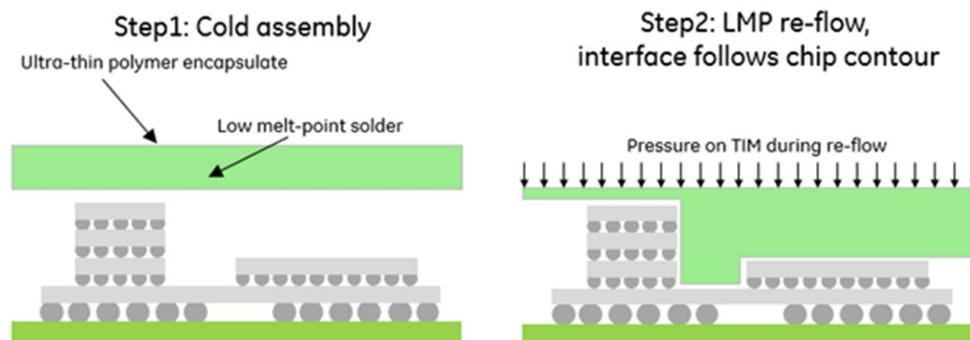


Figure 14 Encapsulated LMP solder thermal interface system with the potential to conform to 3D chip topology

As LMP solder never comes in full contact with the silicon chip, no intermetallics are formed, eliminating the need for barrier coatings and allowing for re-workability. The benefits of such a concept are the ability to compensate for chip height and angle variation, non-planar chip warp, and 3D topologies.

### 2.5.3. Dissimilar heterogeneous component systems solutions

In automotive power conversion, a target of power density increase to  $100 \text{ kW/L}$  is envisioned to be achievable by downsizing the components and managing device heat fluxes on the order of  $1000 \text{ W/cm}^2$ . In addition, increasing the temperature of the wide-bandgap (WBG) devices up to  $250^\circ\text{C}$  is an important enabler towards meeting the goals of downsizing the components. This is a heterogeneous integration thermal management challenge since multiple components with different functionalities and temperature limits (transistors, diodes, capacitors, gate drivers, other passives) need to be packaged within a small volume. Hybrid silicon and WBG technologies and configurations are also of interest – and this is another heterogeneous integration challenge or problem.

Figure 15 shows a schematic of an inverter in which the different components are in a stacked, multilayered, compact configuration. A power module or package incorporating WBG devices is desired that can operate at device junction temperatures up to  $250^\circ\text{C}$ . This will require significant advances in WBG device technology, circuit boards, advanced interface materials and interfaces (likely bonded interfaces), electrical interconnects, encapsulants, electrically-isolating substrates, as well as novel baseplate and heat exchanger materials that can withstand the higher temperatures, give good thermal performance and also have mechanical properties that ensure good reliability for the system (the target numbers have been listed above).

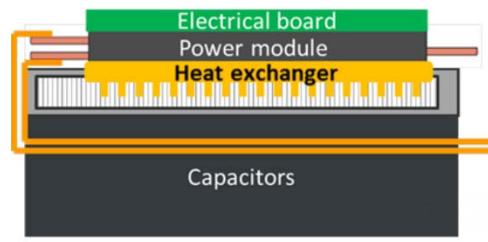


Figure 15: Inverter in a multi-layered board or stack-up configuration

Some components such as gate drivers and capacitors have lower temperature limits – as the state of art stands in terms of materials capabilities – of approximately 125°C or lower. In order to package these passive components – utilizing in-board, stacked 2- or 3-D packaging concepts – in close vicinity of the higher-temperature (e.g. 250°C) semiconductor device or package, advanced thermal management techniques will be required. These include fluid-based cooling technologies, and also concepts such as manipulation of heat flux lines (e.g. metamaterials) so as to divert or block the heat flow to lower-temperature-rated components. There is scope for innovation in the fluid-based cooling technologies; two-phase cooling through passive/pool boiling or forced convective boiling, vapor chambers, heat pipes, and perhaps even fluid-cooling utilizing dielectric fluids within the transistor and diode need to be investigated.

Thermal management of electrical interconnects is an important way to keep the capacitor temperatures within limits [9]. Heat typically flows to the capacitors from the packages/devices/power modules via the electrical interconnects (Figure 15) and thermally managing the interconnects is important for inverter/converter system-level thermal management as well.

Higher voltages are also being considered as a means to downsize the footprint and increase the power density of the power electronics. This will require development of appropriate electrical isolation for the higher voltage, which could have thermal management implications as well. Either the material or the thickness/footprint of the electrical isolation will have to change, and thermal solutions will have to be developed or adjusted accordingly to ensure good thermal performance and reliability as well as to maintain a compact footprint.

#### 2.5.4. Heterogeneous integration thermo-mechanical solutions

Satellite phased-array antenna-in-package solutions offer increased efficiency for higher power applications; however, heat dissipation remains a challenge. Materials in the signal propagation path must be dielectric to minimize signal losses, limiting the applicability of topside cooling strategies. Penetrations in the PCB, and heat sinks with pedestals, have been proposed to bring cooling to the package for these high-power applications [10]. However, in the vacuum environment of space where convective cooling is not available, the need for compliant interface materials that can make contact with an array of discrete components while not suffering performance losses due to the lack of convective enhancement remains a critical need.

In addition to AiP challenges in the space environment, satellites in low earth orbit (LEO) may deploy onboard FPGA system-in-package solutions to enable services such as on-board signal or image processing and 5G internet. Because their orbit is not geosynchronous, these devices typically operate intermittently and dissipate a large amount of heat locally for a short amount of time. Sizing a radiator of the small-LEO satellites for peak heat loads results in excessive area requirements and results in components becoming too cold in non-operating conditions. Furthermore, these components are generally located in the interior of the satellite, without direct access to a radiator. Over time, thermal fatigue also becomes a significant challenge, especially with CTE-mismatched heterogeneous structures. When coupling high-power components to the radiator via heat pipes is not an option, local thermal storage using phase-change materials or other high specific-heat-capacity solutions that minimize required system launch weight may become attractive solutions.

At the same time traditional interfacing strategies for test, including the use of liquid thermal interfaces or dry interfaces, are no longer viable because the varying die heights would result in components with no contact to the thermal control unit (TCU). To fill the gaps between the multiple die and the TCU, durable, compliant, low-compression-set interfaces are required. Silicone-based materials offer the mechanical properties that these applications demand (wide operating temperature range, forgiving compressive mechanics); however, the risk of silicone contamination of the device under test drives the need for new solutions. Aligned, conductive nanosprings, such as carbon nanotube arrays impregnated with polymers, may offer a path forward [11], if they can offer the long-range compressibility (for package die height variations on the order of 100s of microns) as well as low compression

set (to address variation in package-to-package height driven by manufacturing tolerances) that these emerging technologies demand.

### 2.5.5 Heterogeneous integration in harsh environments: conclusion

As with most electronics lifecycles, operation of new technology often starts in highly competitive applications with limited operational life, of which some can be consumer electronics. As these technologies mature and gain more pedigree, more and more application to critical electronics in harsh environments can be considered. This study identified three areas of technology research that can be focused on to further aide this progression. These are the development of 3D thermal interface technology, application of CTE-matched heat spreaders, and active transient thermal hot spot management. It is anticipated that with time and development support, these technologies will further advance, allowing one day for safe and reliable operation of heterogeneously integrated electronics in the transportation, industrial and other safety-critical systems of tomorrow.

### 2.6 Thermal challenges in mobile platforms

The shrinking form factor of mobile electronic devices in conjunction with significantly increasing performance and functionality have resulted in substantial thermal challenges. Nelson and Galloway [12] report reduction in thickness of Smart Phones from about 25 mm to less than 10 mm while power has increased from below 3 W to about 7 W for the latest devices. The total number of packages inside such smartphones have also grown over several product generations to about 70 [12]. Figure 16 [12] shows the external surface temperature profile for a commercial smartphone and illustrates the presence of a significant hotspot at the location of the processor.

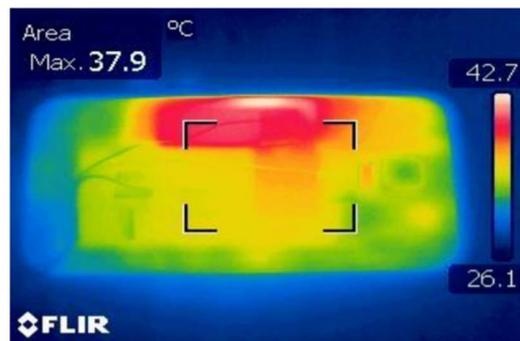


Figure 16: Temperature contour data for the external surface of a smartphone [12]

Steady state and transient thermal characteristics and spreading from hot spots primarily through conduction (copper, PCB, silicon, TIM) are reported to be key drivers for thermal design [12] with dynamic thermally-aware power throttling being an effective control technique for thermal management. Figure 17 provides an inside view of a State-of-the-Art Smart Phone [12] and shows a micro heat pipe spreader attached to multiple devices and packages to promote heat spreading from multiple heat sources. Graphite sheets and copper “straps” have also been harnessed to this task.

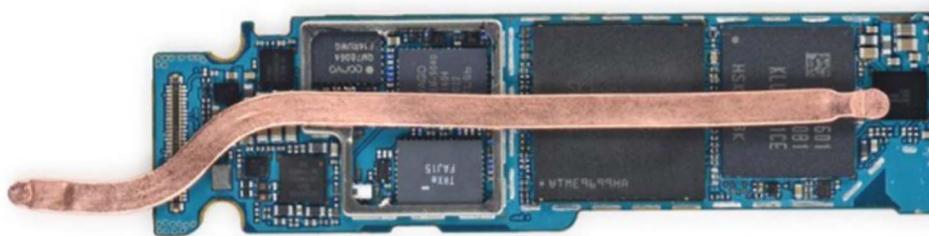


Figure 17: Micro heat pipe in a modern smartphone [10]

### 2.7 Thermal challenges in voltage regulators

Scaling trends and device refinements indicate gradual and steady transition to lower-threshold voltage regulator devices, driving up package-level current to several 100s of amps. Ohmic losses on power delivery connections tend to dominate the power dissipation. However, local decoupling needs exacerbate the challenges associated with higher voltage DC distribution and down-conversion to required DC levels with regulators on/near die (and on-die) within the package. Thus, one of the power devices that can be part of a heterogeneously integrated package is a VR (Voltage Regulator) such as what is depicted in Figure 18. Such power devices inside the package are potential hot spots and

create a significant cooling imperative. For example, even at 95% efficiency, a 200W VR will dissipate 10 Watts, mostly within the power switching devices with a small footprint inside the package.

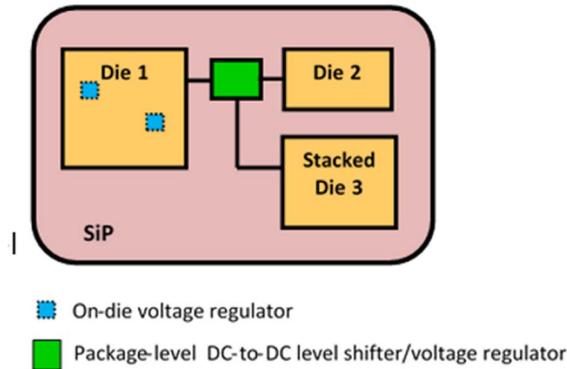


Figure 18: Package level DC to DC VR schematic

### 3.0 Advanced Technologies and Research Innovations

#### 3.1 Thermal interface materials

Electronic device performance is constantly improving, but with this evolution comes greater power consumption and heat generation. It is imperative to effectively transfer heat from semiconducting materials (any hotspot surfaces in the computing system: e.g., integrated circuit chips, a central processing unit, and a graphics processing unit) to the metals used for heat spreading and exchange through the interface materials. Thermal interface materials (TIMs) provide a low-resistance thermal pathway between two surfaces by filling the interfacial volume created when two microscopically rough surfaces are in contact [13, 14]. An ideal TIM is both thermally conductive, to facilitate heat transfer across the interface, and mechanically compliant, to conform to the surface roughness and to maintain the interface’s integrity despite thermomechanical stresses imposed by temperature gradients, thermal cycling, and thermal aging. However, these two properties often scale dichotomously, where high thermal conductivity materials are typically dense and stiff, while soft materials are generally poor thermal conductors [13-15]. As a result, most commercially available TIMs are either thermally conductive (e.g., solder) or mechanically compliant (e.g., thermal paste), but rarely both.

Two strategies are commonly used to create TIM composites that pursue the combination of high thermal conductivity and mechanical flexibility, as illustrated in Figure 19.

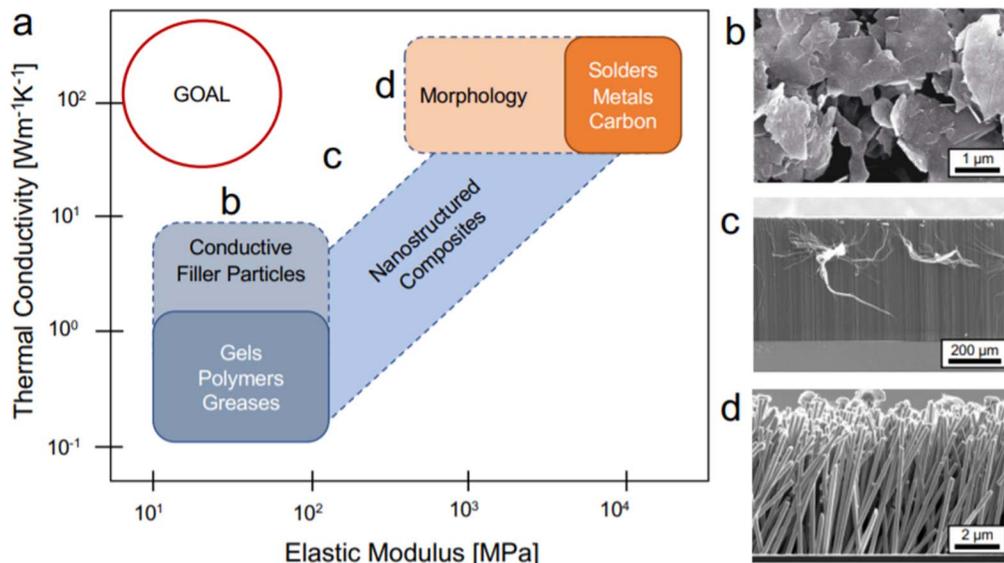


Figure 19 (a) Two common strategies can be employed to create high-performance TIM composites [16], (b) an example of graphene-polymer composite [17], (c) vertically grown nanotubes [18-19], (d) vertically electrodeposited nanowires [16, 21].

One strategy is to begin with an intrinsically soft material and add fillers to increase the thermal conductivity, such as an example of graphene-polymer composite in Figure 19 (b). The second strategy is to form a nanostructure from an intrinsically conductive material into a mechanically compliant morphology, such as Figure 19(c) vertically grown nanotubes [18-19], and (d) vertically electrodeposited nanowires [16, 21].

For the first strategy described above, the method of manufacture would be to begin with an intrinsically soft material, such as a polymer, and add conductive fillers (e.g., metal nanoparticles, carbon nanoparticles, and graphene nanomaterials) to increase the effective thermal conductivity by creating a conduction network [22].

For the second strategy, the fabrication process would be to build the nanostructured conductive material (e.g., metals and graphite) into a mechanically compliant morphology. Aligned arrays of continuous, conductive elements improve the effective thermal conductivity by providing out-of-plane heat transfer pathways. Recent approaches have focused on synthesizing carbon nanotubes (CNTs) into vertically aligned arrays [18-19]. However, the CNT arrays show lower thermal conductivity values (of 10-50 W/m-K) than theoretical estimates, due to the material's uncontrollable morphology during the growth process, nanotube-nanotube contact resistances, and defects [18-19]. More recent work has explored the use of templated electrodeposition to design metal architectures (i.e., aligned metal nanowires [16], porous metal structures [23] and nanosprings [24]) in a controlled manner, in order to precisely engineer mechanically compliant TIMs. In addition to the stated strategies, an etched heat sink is another strategy to make an available TIM more effective.

To address various challenges of current TIMs, a new class of compliant and ultrathin TIMs can be developed consisting of copper nanosprings embedded in a polymer layer by using the templated electrodeposition method. The proposed nanosprings with 200 nm diameter and > 25% volume fraction will result in an effective thermal conductivity of 100 W/m-K, leading to conduction thermal resistances smaller than  $< 0.5 \text{ mm}^2 \text{ K/W}$  with  $< 50 \text{ }\mu\text{m}$  thickness. The installation of elastic nanosprings can help to minimize the boundary thermal resistances between nanostructures and the surface, by accounting for a surface roughness tolerance, resulting in a value of  $< 0.5 \text{ mm}^2 \text{ K/W}$ . The temperature- and pressure-dependent thermal characterizations for varying TIM morphology and bonding methods can be followed to advance the understanding of the material's structure-related properties [21].

Besides CNT arrays and metal nanosprings that yield anisotropic thermal conductivity of the TIM, progress as illustrated in Figure 20 [25] is being made in the synthesis of porous continuous ultrathin graphitic foam (UGF) structures and in using them to achieve an isotropically high thermal conductivity. Methane chemical vapor deposition (CVD) on sacrificial sintered nickel powder has yielded UGF structures with a micrometer-scale pore size, macroscale lateral dimension, and effective thermal conductivity approaching 20 W/m-K at a porosity larger than 90%. The nickel powder can be recycled with an electrochemical etching and deposition process to lower the manufacturing cost. Besides electrically conducting UGF, electrically insulating, semi-transparent, thermally conducting porous foam architectures of hexagonal boron nitride (h-BN) can also be grown with a similar CVD process. Both the UGFs and h-BN foams can be explored further to serve either as high-thermal conductivity fillers of polymeric TIMs or polymeric substrates for future-generation flexible electronics.

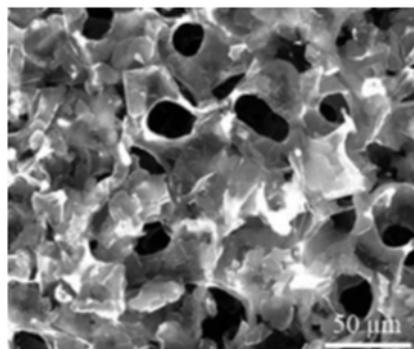


Figure 20: SEM image of a porous ultrathin graphitic foam structure [25]

Oftentimes air voids formed in between the conducting surfaces hinder heat transfer and lead to increased temperature. Recently Gamal et al [20] proposed a TIM employing a surface-modified etched heatsink with a screen-printable phase-change material-based TIM. The TIM as a standalone system has a thermal conductivity of 5.4 W/mK, and flows at temperature above 45°C without changing phase. This specialty TIM was combined with a surface-etched heat sink as shown in Figure 21 [20], and an enhanced thermal conductivity of 22W/mK at 75 microns thick was achieved. The advantage of using the engineered Phase Change Material TIM with etched surface is shown

in Figure 22. From the test results it is clear that there is a 15°C and 6°C reduction in chip surface temperature compared to Indium and Graphite (both higher performance materials), respectively, for the cases considered.

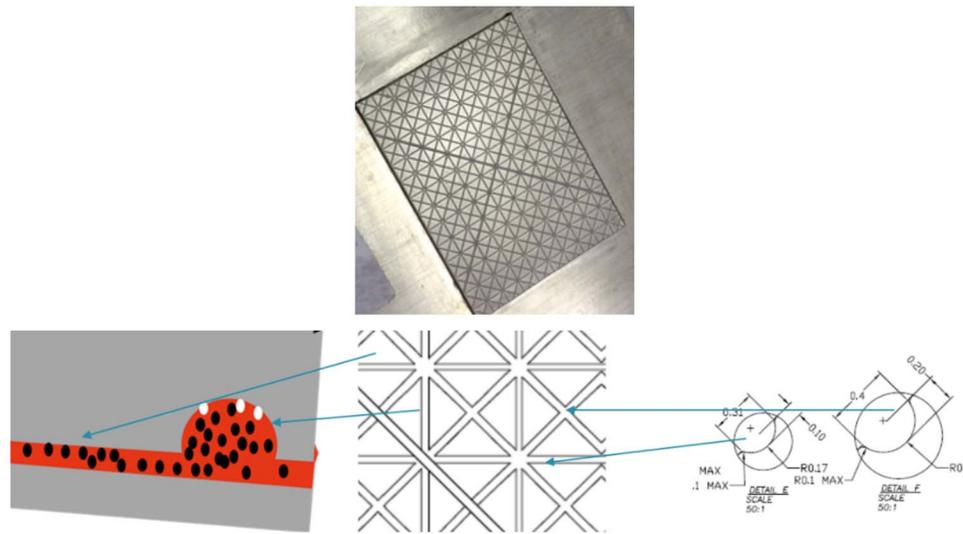


Figure 21: Textured (etched) surface having multiple grooves as a new strategy to improve interfacial conductivity [20]

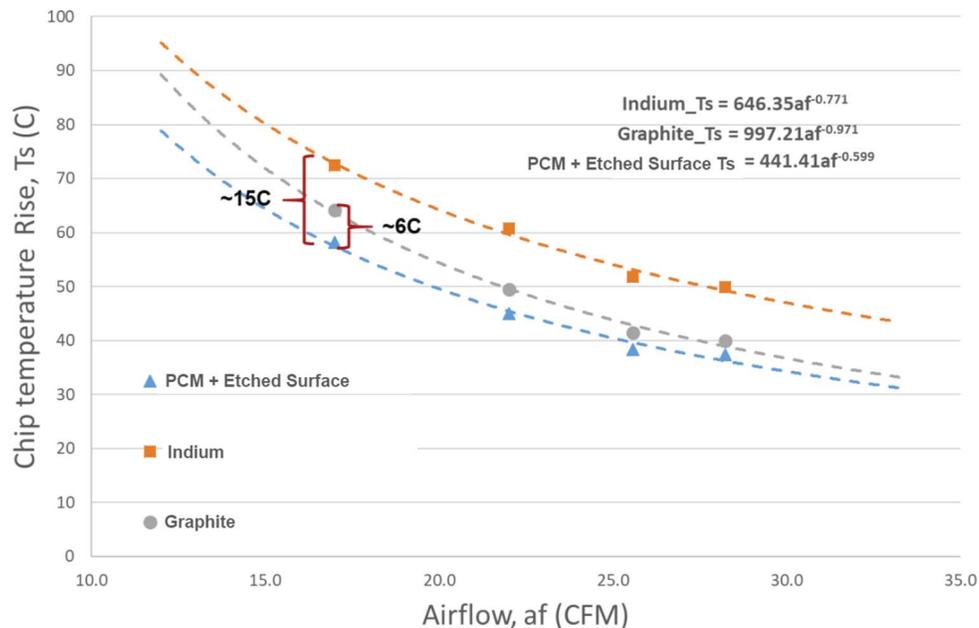


Figure 22: Plot showing the chip temperature rise vs air flow for various TIMs tested on server with Thermal Design Power for the chip 130W

### 3.2 System thermal limits for HPC multi-chip modules

Thermal management of multi-chip heterogeneously integrated systems poses additional constraints and limitations beyond those for single-chip modules or vertically integrated systems. Applications related to HPC systems tend to be high power and have strict reliability and availability requirements. The major thermal concerns are hot spot junction temperatures and transient excursions. They may also have multi-chip modules in close proximity and possibly with different thermal requirements, non-uniform device heights off of the substrate, hot spots on different devices, and transient excursions in workload and power. The emphasis here is on HPC applications, and the intent is to summarize either commercially available or demonstrated system solutions, cooling limits for air, single-phase liquid cooling, or two-phase cooling. Establishing heat flux limits is not a simple matter since the maximum possible heat flux limit depends on many application-specific factors, including the number of devices, the spatial device power distribution (i.e. power non-uniformity), the allowable junction temperature, the allowable pressure drop, the volume and weight limitations for the cooling solution and acoustic constraints, among others.

### 3.2.1 Air cooled heat sinks

For multi-chip modules, a typical high heat flux thermal solution may look like the schematic diagram shown in Figure 23 (adapted from [26]). Air-cooled heat sink modules are known to be highly reliable thermal management solutions. However, the reliability is at the expense of a large areal and volumetric footprint due to the inferior thermal properties of air, especially the thermal conductivity and low density. The air-cooled heat sink module system comprises a sequential arrangement of thermal interface material (TIM), spreader and heat sink. The performance of the heat sink is largely a function of heat spreading. To augment the cooling limit of an air-cooled heat sink, a high-performing heat sink with vapor chamber and high-conducting TIM must be combined. The limit on the maximum air flow is dictated by the acoustic constraints and the allowable pressure drop in the system. One example of estimating air-cooling limits assumes a uniform heat flux at the devices; the maximum possible device heat flux is estimated to be about  $84 \text{ W/cm}^2$  [27] for a specific set of assumptions. However, more recent studies [28] have utilized the limit on minimum inlet air temperature as dictated by the Telcordia GR63 standard and ASHRAE for telecom applications ( $55^\circ\text{C}$ ) and data center applications ( $45^\circ\text{C}$ ) respectively, as well as including heat spreading assumptions (vs uniform heat flux). Within these practical limitations, the efficient air-cooled module demands a high-conductive TIM with a thin bond line and a highly efficient vapor chamber (with high dry-out heat flux) that has a thin profile. A recent study [28] experimentally demonstrated a unit area junction to ambient air-cooled heat sink limit of  $0.8^\circ\text{C}\text{-cm}^2/\text{W}$  thermal resistance (this is a thermal resistance commensurate with  $0.8^\circ\text{C}$  temperature rise across  $1 \text{ cm}^2$  of area with  $1\text{W}$  of heat dissipation). The heat sink housed a heat pipe in addition to the vapor chamber, to efficiently move the heat to the lateral regions of heat sink, thereby making fins more efficient and increasing the efficiency of the overall heat sink. Higher heat flux levels may be possible through further optimization of the heat sink geometry and/or vapor chamber design and the TIM thermal resistance, or by relaxing the practical constraints such as the allowable heat sink volume or air flow.

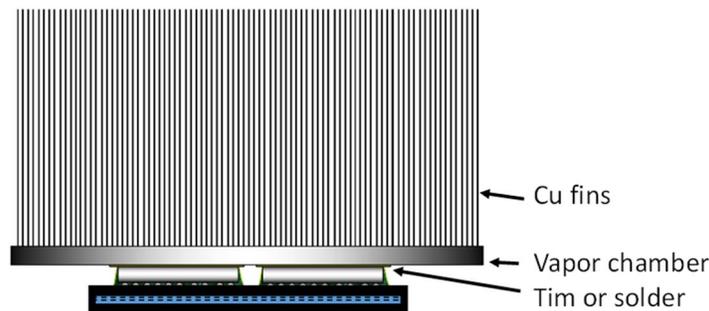


Figure 23: Figure adapted from [23] showing a schematic diagram of a heterogeneously integrated multi-chip module with a Thermal Interface Material (TIM) or solder attachment and a vapor chamber.

In order to establish the air-cooling limit for differing physical form factor heat sink solutions, an in-house experimental series was performed to arrive at a global equation. This equation describes the junction to incoming air resistance impedance as a function of the flow parameters. The inlet air temperature was chosen based on the Telcordia GR63 standard and ASHRAE for telecom application ( $55^\circ\text{C}$ ) and data center application ( $45^\circ\text{C}$ ) respectively. Tests were also conducted for different thermal solutions that can be used in either 1U (1.75" industry standard server) to 4U (7" tall industry standard server) data center server applications. The in-house test established another level of confidence on the air-cooling limit in Figure 24 and Table 1, which is around  $40\text{-}53 \text{ W/cm}^2$  for a thermal budget of around  $45^\circ\text{C}\text{-}50^\circ\text{C}$ . Using the test result data points, the following equation was established [28]:

$$R_{JA}^* = 192.8 (V_{HS}/A_{af})^{-1.258} F^{-0.30}$$

where:

$R_{JA}$  is Unit Area Junction to air resistance in  $^\circ\text{C cm}^2/\text{W}$

$F$  is flow speed in LFM (Linear Feet Per Minute)

$V_{HS}$  is heat sink volume in  $\text{cm}^3$

$A_{af}$  Air flow approaching area in  $\text{cm}^2$

By employing the above equation, for both the 1U and 4U standard servers, respectively, the  $R_{JA}$  was  $1.22^\circ\text{C cm}^2/\text{W}$  under a constrained thermal budget of  $50^\circ\text{C}$  from junction to inlet air (ambient). It was also noted that the designer should account for the spreading resistance where it is varied based on the structure of the package and/or the power map on the chip. Figure 24 shows reported data from [27] and the latest air and liquid cooling limits [28,

29]. Refai Ahmed et al. [28] developed the curves of thermal resistances based on a 1D resistance model. The objective of the curves of thermal resistances in Figure 24 was to give an example of an engineering cost reduction practice. This engineering practice is to present, at a target thermal resistance, a solution that has a high-performance heat sink solution and low performance thermal interface materials, or vice versa. This was given as guidance for the practitioner who needs to do cost optimization. However, to extend air cooling's limit, the practitioner must use both a high-performance heat sink and a low-resistance TIM similar to the work carried out in this study. Figure 24 was constructed based on the total available system resistance from junction to ambient. Resistance from junction to inlet air (ambient) was varied from 0.5 to 0.05°C/W. From the known TIM and spreading resistance, the available heat sink thermal resistance is determined and plotted.

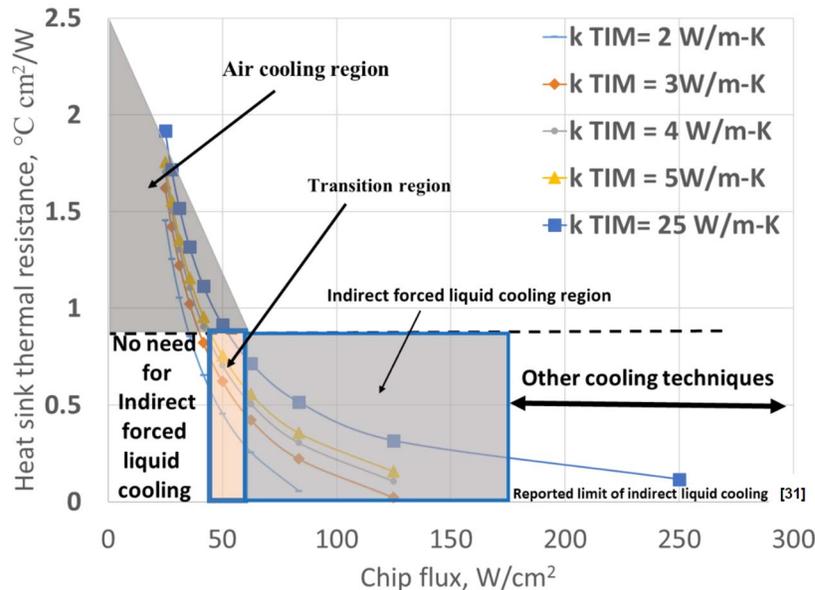


Figure 24: Heatsink thermal impedance,  $RS-A$ , versus chip heat flux with regimes marked for different cooling technologies [28]

### 3.2.2 Liquid and two-phase cooling

There are several approaches to liquid cooling of electronic systems. These can be broadly classified into indirect and direct liquid cooling. In direct (immersion) systems the coolant comes into direct contact with the devices and therefore the cooling material needs to be a dielectric. This could exclude water as the cooling medium and is therefore a significant limitation in terms of achieving very high heat flux levels. To make water as a medium work for such configurations would require electrically insulating technology (e.g. plated ground planes, sufficiently thicker chips) to isolate the devices from the water. It is also a challenge to have the cooling fluid come into direct contact with the electronic devices, since any organic materials in the system pose the risk of degradation and leaching into the coolant over time. The advantages of direct liquid cooling are primarily the reduction of thermal resistances such as those of heat spreaders and TIMs. Chip-scale cold plate designs with fluorocarbon cooling can be used with heat flux levels of up to 100 to 300 W/cm<sup>2</sup> [30-33]. The range depends on specific design and system considerations. Even higher heat flux levels may be achieved if cost and space limitations are expanded.

Indirect cooling is distinguished by the presence of a physical barrier between the devices and the coolant, thereby eliminating concerns about chemical interactions between the coolant and the devices and also making the usage of water (with additives) a possibility. Direct and indirect liquid cooling, both single-phase and two-phase, can be used, with distinct advantages and disadvantages to each approach. Single-phase cooling systems are easier to design and operate. However, when single-phase systems are used to cool multiple devices in series, the incoming coolant keeps rising in temperature, which may be a limitation depending on the application. Two-phase systems, if well controlled, have the advantage of operating at nearly uniform temperatures even when used serially to cool multiple modules. Indirect water cooling can achieve significantly higher heat flux levels of up to 300 to 450 W/cm<sup>2</sup> for chip-scale water jet impingement solutions.

A recent study [29] established a limit of 0.1°K cm<sup>2</sup>/W (based on chip area) for a separable remotely liquid cooled module. The optimal values obtained for the fin thickness are within the range of 0.25-0.3 mm and the optimal channel width was close to 0.4 mm. The study established a limit of Unit Area Thermal Resistance  $R_{JA} = 0.35^{\circ}\text{C cm}^2/\text{W}$  (based on chip area) corresponding to a chip heat flux of 170 W/cm<sup>2</sup> (marked in Figure 24) for a remote

cooled module for liquid cooling employing parallel channels operating below a constrained pressure drop value of 10kPa. From the series of detailed experiments, a correlation for Unit Area Thermal Resistance from junction to inlet fluid ambient,  $R_{JA}$ , was determined as follows:

$$R_{JA} = 0.47 \times FR^{-0.276}$$

where FR is the liquid flow rate in liters per minute, and the equation is valid between  $0.3 < FR < 1.5$

Table 1, including data adapted from [26] and more recent published data, provides a summary of cooling limits for air, FC and water as well as the thermal resistance (air cooled module) and thermal insulation (water and FC).

*Table 1: Comparison of system cooling capabilities*

Thermal solution	High end chip scale heat flux (based on chip area)	$R_{int}$ Chip to heat sink/cold plate (based on chip area)	$R_{ext}$ heatsink/cold plate (based on chip area)
Advanced multi-chip air cooling with integrated vapor chamber	0.85 W/mm <sup>2</sup> (uniform heat flux)	Vapor chamber 0.267 °C/W (w/ Grease) 0.2 °C/W (w/ solder) for Chip to fins	0.1 °C/W for Al .069 °C/W for Cu [23]
	0.55 W/mm <sup>2</sup> (thermal budget 50 °C) [28] chip area = 4 cm <sup>2</sup>	Vapor chamber 0.03°C cm <sup>2</sup> /W [34] 0.0375 °C cm <sup>2</sup> /W (TIM 20W/mK and 75 microns thick) [20]	0.8 °C cm <sup>2</sup> /W for Aluminum [28]
Water cooled separable module level cold plate	2.50 W/mm <sup>2</sup> (uniform heat flux)	0.35 cm <sup>2</sup> K/W Lid +TIM	0.128 cm <sup>2</sup> K/W
	1.70 W/mm <sup>2</sup> (thermal budget 50° C) Chip area = 4 cm <sup>2</sup> , heat sink base area = 16 cm <sup>2</sup> [29]	0.2 °C cm <sup>2</sup> /W spreading on 3mm thick Cu spreader + 0.0375 °C cm <sup>2</sup> /W (TIM 20W/mK and 75 microns thick) [29]	0.1 °C cm <sup>2</sup> /W [29]
Water jet impingement [35]	4.6 W/mm <sup>2</sup>	N/A	0.025 cm <sup>2</sup> K/W
Water immersion cooling [36]	5.62 W/mm <sup>2</sup> for a device area of 12.5 mm <sup>2</sup> at Thermal budget of 100 °C Fluid temperature = 25 °C (Power electronics)	N/A	0.18 °C cm <sup>2</sup> /W
Two phase (dielectric) immersion cooling [36]	1 W/mm <sup>2</sup> for a device area of 12.5 mm <sup>2</sup> at Thermal budget of 50 °C Fluid temperature = 25 °C (Power electronics)	N/A	0.5 °C cm <sup>2</sup> /W
Micro channels in the device with single phase water flow [37]	About 7.90 W/m <sup>2</sup> 71 °C temp rise, Chip size = 1 cm <sup>2</sup>	N/A	0.09 °C cm <sup>2</sup> /W
Micro channels in the device with two phase [38]	About 10.20 W/mm <sup>2</sup> (uniform heat flux) 58 °C temp rise Chip size = 0.25cm <sup>2</sup>	N/A	0.056 °C cm <sup>2</sup> /W

Water has a high thermal conductivity and specific heat capacity and therefore outperforms FCs. Compared to the use of indirect liquid cooled cold plates that are attached (mounted) on a chip using a thermal interface material (TIM), reduction in thermal resistance would require etching microchannels or fins in the silicon directly, as demonstrated by Tuckerman and Pease [37] who demonstrated a heat flux of 790 W/cm<sup>2</sup> at a maximum temperature rise of 71°C. Although this approach has not been commercialized, it may be considered to be close to the best achievable ultimate heat flux level for silicon devices with longitudinal microchannels. Manifold-based microchannel configurations discussed subsequently in this chapter outperform single-pass longitudinal arrays. This would be approximately true for 3D stacks with cooling liquid through the stack as well. For this case, the internal thermal resistance is 0 and the external resistance will be a function of the microchannel design.

### 3.2.3 Summary of System Cooling Limits

Establishing the pragmatic limits on the most promising system cooling technologies in combination with the best thermal interface materials becomes very important for understanding the thermal limits. A nominal summary overview for the TIM and system (air and liquid) cooling technologies discussed in the previous two sections has been combined to yield the analyses displayed in Figure 25.

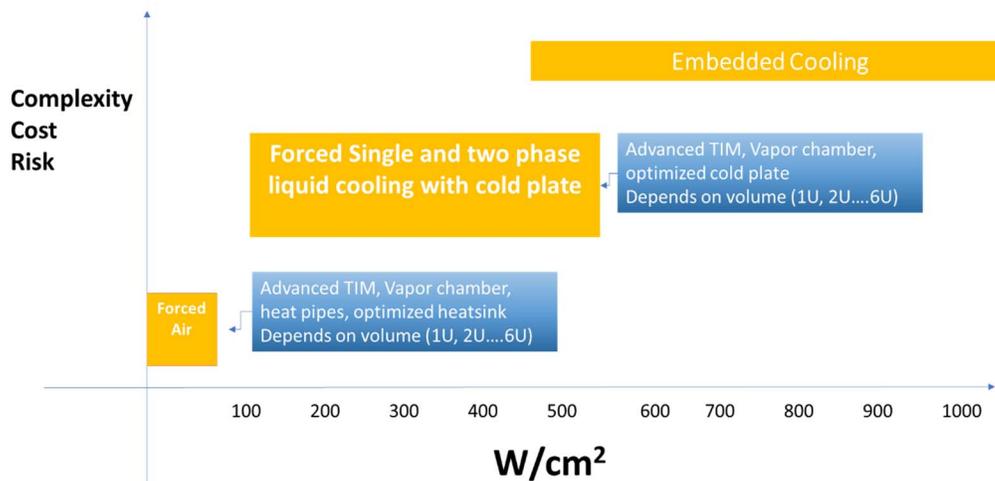


Figure 25: Nominal summary overview for the TIM and system (air and liquid) cooling technologies

### 3.3 Embedded liquid cooling of chips and chip stacks

Embedded cooling [39] represents a third generation (“Gen3”) thermal management technology for electronic circuits and was the focus of DARPA’s recent Near Junction Thermal Transport [40] and Intra/Interchip Enhanced Cooling [41-42] thermal packaging programs. The DARPA Near-Junction Thermal Transport (NJTT) program, initiated in 2011, was the first program to develop thermal management for the region within 100µm of the electrical junction of a GaN transistor, to enable heat fluxes of greater than 10 kW/cm<sup>2</sup> while maintaining reliable junction temperatures. Through technology developed in this program, specifically the transfer of GaN epitaxy to high thermal conductivity diamond, the power handling capability of GaN HEMT devices was increased by greater than a factor of 3 [43-48].

While the NJTT program made significant gains using high thermal conductivity substrates to spread the heat close to the junction, it did not address the next link in the thermal resistance chain, i.e. extracting the heat from the diamond and transferring that dissipated heat to an available coolant. The DARPA ICECool program [49], which began in 2013 and is now nearing completion, combined embedded microfluidics with high thermal conductivity substrates to reduce the thermal resistances in the entire package. The successful demonstration of embedded cooling by three of the participating research teams is described in this section.

#### 3.3.1 2D chip direct contact liquid cooling with hierarchical manifolds

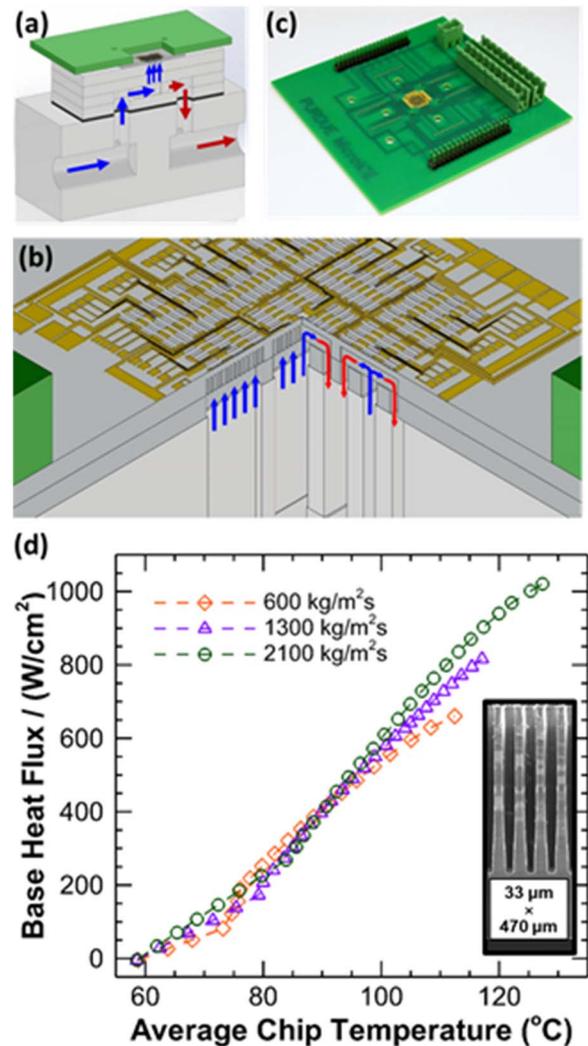
Electronic devices have traditionally been cooled through the attachment of separate heat sinks. In this ‘remote cooling’ architecture, the performance of the thermal management solution has become increasingly governed by interfacial, conduction, and spreading thermal resistances incurred by the package and the mechanism of heat sink attach. Trends in heterogeneous integration call for the development of transformative cooling strategies that embed the thermal management solution directly within the device to alleviate hotspots. ‘Intrachip cooling’ strategies deploy coolant channels directly in the electronic device, eliminating interfacial resistances but leaving little material

thickness available for heat spreading; this exposes the embedded microfluidic heat sink directly to the high heat fluxes generated from the device. Dielectric working fluids are preferred for such systems because they minimize the threat for electrical shorting.

As part of the DARPA ICECool Fundamentals program [49], researchers at Purdue University have demonstrated dissipation heat fluxes exceeding  $1000 \text{ W/cm}^2$  by feeding an array of high-aspect-ratio, intrachip microchannel heat sinks in parallel with a dielectric coolant via a manifold for fluid distribution. The fabrication of suitably high aspect ratio microchannels etched into the silicon test chip provides the necessary surface area enhancement to dissipate high heat fluxes at an allowable surface temperature rise; the microchannels are imbedded directly into the heated substrate to reduce the parasitic thermal resistances due to contact and conduction resistances. Parallelization of the flow across an array of short-flow-length microchannel heat sink elements serves to minimize the pressure drop across the heat sink. This allows for flow boiling operation in the microscale channels, which would otherwise be prohibitive due to pressure drop constraints.

A hierarchical manifold microchannel heat sink array test vehicle, with all flow distribution components heterogeneously integrated, was fabricated to demonstrate thermal and hydraulic performance of this technology (Figure 26, [50-51]). A silicon chip, with resistive heaters and local temperature sensors fabricated directly on the opposite face, is cooled by a  $3 \times 3$  array of microchannel heat sinks using dielectric HFE-7100. The heat sink performance is characterized over a range of channel mass fluxes and channel geometries. At a mass flux of  $2100 \text{ kg/m}^2\text{s}$  and for microchannel channel cross sections with widths of  $33 \mu\text{m}$  and depths of  $470 \mu\text{m}$ , uniform heat fluxes up to  $1020 \text{ W/cm}^2$  are dissipated over a  $5 \text{ mm} \times 5 \text{ mm}$  heated area, at chip temperatures less than  $69^\circ\text{C}$  above the fluid inlet and at pressure drops less than  $120 \text{ kPa}$ . Experiments were also conducted for heat fluxes generated up to  $2,700 \text{ W/cm}^2$  from a  $200 \mu\text{m} \times 200 \mu\text{m}$  hotspot heater overlaid on the background heat flux. This work demonstrates the fabrication, integration, and characterization of hierarchical manifold microchannel heat sinks operating in the two-phase regime.

Figure 26: Hierarchical manifold microchannel heat sink array: (a) three-dimension drawing of the test vehicle with a half-symmetry section removed and fluid inlets (blue) and outlets (red) shown; (b) zoomed-in view of the test vehicle with a quarter-symmetry section removed showing the fluid flow paths in the test chip; and (c) photograph of the test chip mounted to the PCB with heaters and sensors face up. (d) Experimental characterization of the base heat flux as a function of average chip temperature for a heat sink with  $33 \mu\text{m} \times 470 \mu\text{m}$  cross section microchannels at three different mass fluxes.



### 3.3.2 Extreme heat flux micro-cooler for direct contact 2D chip liquid cooling

Thermal-power challenges and increasingly expensive energy demands pose threats to the historical rate of increase in processor [55] and power electronics performance. Energy-efficient computing [56] and heterogeneous integration [57] promise substantial reduction in energy demand for emerging and growing computing needs. However, these conflicting trends have resulted in a substantial increase in both heat flux  $>350 \text{ W/cm}^2$  and power density, which reduced the efficacy of conventional cooling technology solutions.

Figure 27 depicts the thermal resistance ( $\text{cm}^2\text{-K/W}$ ) versus chip area for the state-of-the-art high heat flux ( $\sim 1 \text{ kW/cm}^2$ ) cooling technologies, which indicates it is extremely challenging to reject heat from large-area devices using single/two-phase  $\mu$ -channel coolers (2D and 3D manifold) [52-62]. This originates for the presence of long traverse paths for liquid delivery and vapor extraction as well as the finite thickness of the liquid evaporation film (or thermal boundary layer), which result in a large pressure drop and increased junction temperature (or thermal resistance); note the trend shown by the yellow band, depicted in Figure 27.

A potential solution could be in the form of an Extreme Heat Flux  $\mu$ -Cooler (EHF $\mu$ -C), depicted in Figure 28, that simply “scales up” a high heat-flux thin-wicking structure, either copper inverse opals (CIOs) [52] or silicon pin fin arrays, to a large area EHF $\mu$ -C using 3D manifold liquid delivery and vapor extraction conduits [53-54]. Two of the state-of-the-art technologies [52-53] have been developed at Stanford [52-55,61] since 2010 through collaborations with Toyota, Ford, Google and IBM, and with funding from DARPA’s IceCool program and the NSF-Center for Power Optimization of Electro-Thermal Systems (POETS). Currently, Stanford is working on the next generation of high performance EHF $\mu$ -Coolers (ARPA-e funded). The proposed EHF $\mu$ -C (Fig. 28) is produced by bonding a 3D-manifold to a silicon substrate electro-plated with a copper inverse opal (CIO) wick. Liquid channels deliver liquid to the CIO wick, where it is pulled in by capillary forces, and evaporates due to the high heat flux  $\sim 1 \text{ kW/cm}^2$  at the substrate.

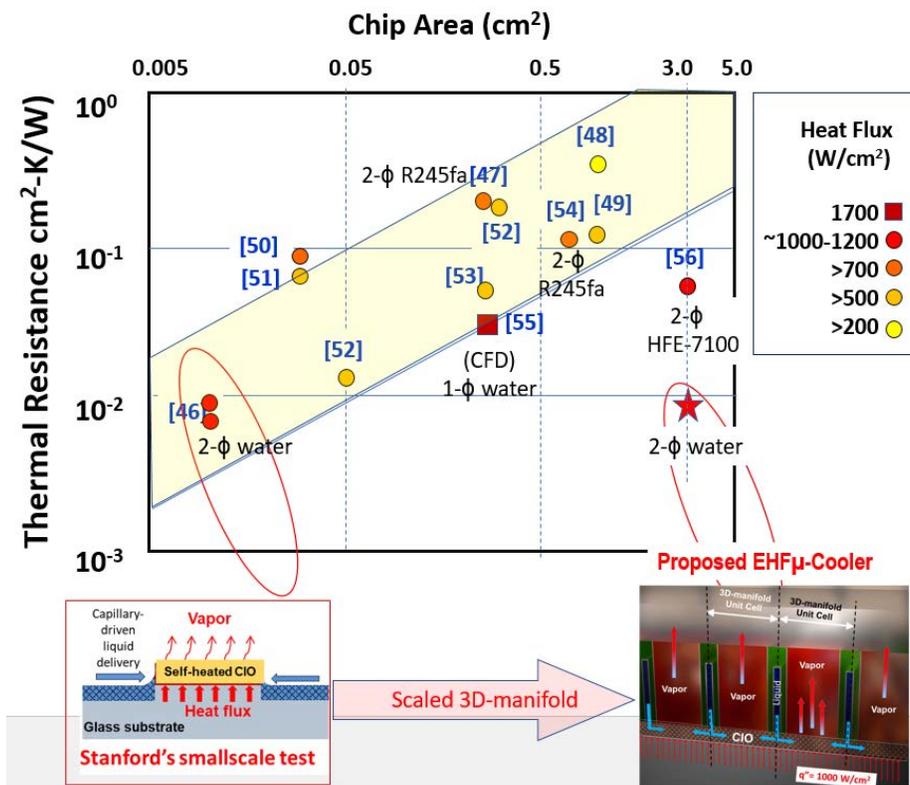


Figure 27: The expected performance targets for EHF $\mu$ -Cooler, and that of the state-of-the-art devices including Stanford’s previous work [52-53]. Critical heat flux levels are from 200 to 1000  $\text{W/cm}^2$ . The EHF $\mu$ -Cooler represents more than  $10\times$  reduction in thermal resistance, an unprecedented CHF  $>1 \text{ kW/cm}^2$  for water as working fluid, and can be scaled up to large areas  $>10 \text{ cm}^2$ .

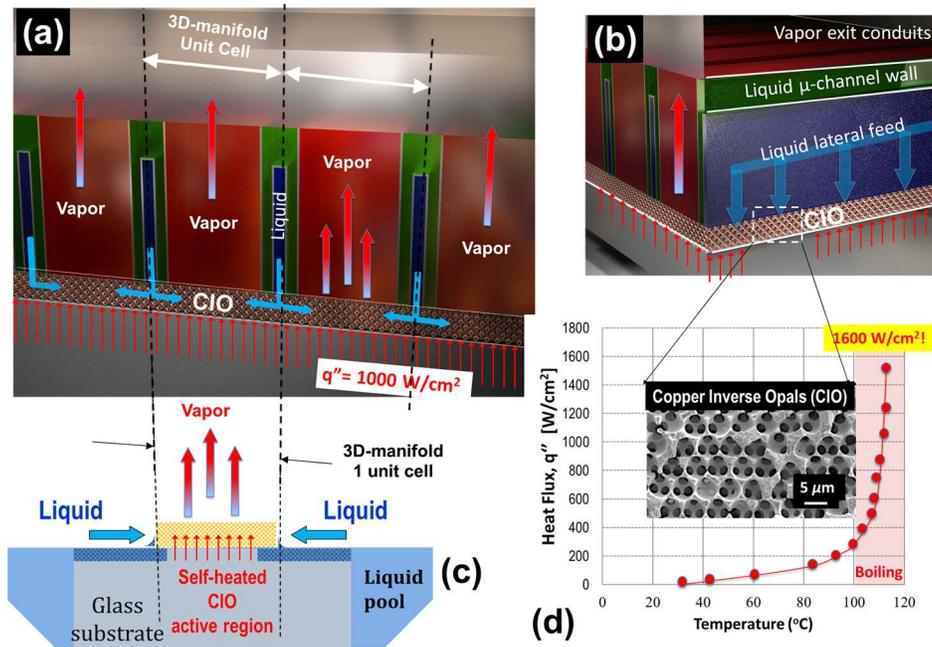


Figure 28: Schematic of the EHF $\mu$ -C concept: (a,b) front & sides cross section views. The EHF $\mu$ -Cooler is produced by bonding a 3D-manifold to a silicon substrate electro-plated with a copper inverse opal (CIO) wick. Liquid channels deliver liquid to the CIO wick, where it is pulled in by capillary forces, and evaporates due to the high heat flux  $\sim 1\text{kW}/\text{cm}^2$  at the substrate. (c) A schematic of Stanford’s “small-scale” test structure [52], (d) capable of removing  $1600\text{ W}/\text{cm}^2$ , inset SEM image of CIO with pore diameter  $\sim 5\ \mu\text{m}$ .

### 3.3.3 Embedded interlayer cooling for chip stacks with two-phase refrigerant

Embedded interlayer cooling technology provides a solution for cooling 3D chip stacks where a heat sink or cold plate is inadequate for thermal management of 3D stacking of high-power chips because of their inability to cool chips in the middle and bottom of the stack. This chip-embedded cooling technology circumvents that problem by pumping a heat-extracting dielectric fluid into microscopic gaps, some no wider than a single strand of hair ( $\sim 100\ \mu\text{m}$ ), between the chips at any level of the stack. The dielectric fluid used can come into contact with electrical connections, so is not limited to one part of a chip or stack. This ability benefits chip stacks in terms of materials and architecture, such as putting memory and accelerator chips on top of high-power chips in the stack as shown in Figure 29 which includes a fluid port to deliver fluid between the stacked dies.

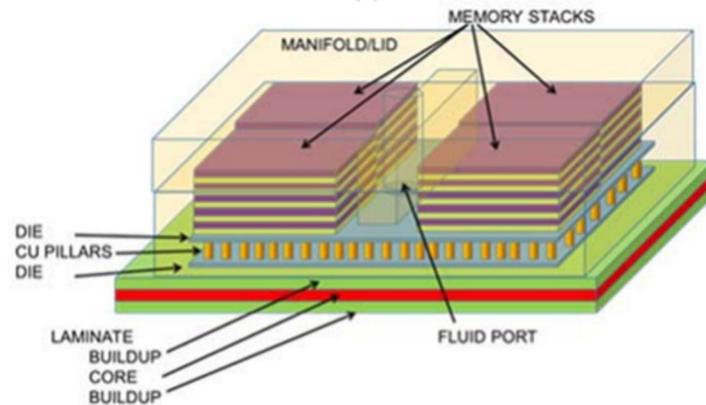


Figure 29: High power 3D package with interlayer cooling [63]

The coolant is pumped into the chips, where it removes the heat from the chip by boiling from liquid-phase to vapor-phase. It then re-condenses, dumping the heat to the ambient environment where the process begins again, as shown in Figure 30. As this cooling system doesn’t need a compressor, it can operate at much lower power compared to typical refrigeration systems.



Two-phase flow boiling has long been proposed as a potential method for cooling high-performance computer systems. A large body of work investigating and developing technologies appropriate for cooling electronics with two phase flow boiling in parallel micro/mini-channels exists, but parallel channel two-phase flow is challenged by instability issues, particularly with non-uniform power maps. This technique utilizes a significantly different approach to embedded cooling. Rather than moving coolant from one edge of the die to the other through long parallel channels, a dielectric coolant (R1234ze or similar) is fed in at the center of the die, moves through radially expanding channels, and exits at the edges of the die. This approach provides better energy efficiency and maximum critical heat flux with the resulting reduced flow path. The cooling capability was demonstrated on a specially constructed thermal test vehicle (Figure 31 a,b,c,d) designed to mimic the heat generation capability of real microprocessors without requiring actual transistor-based circuitry. In these studies, power densities of  $350 \text{ W/cm}^2$  within an area measuring  $3.6 \text{ mm} \times 4.8 \text{ mm}$  representing a microprocessor core and  $200 \mu\text{m} \times 200 \mu\text{m}$  hot-spot power levels of more than  $2 \text{ kW/cm}^2$  were shown to be effectively cooled with chip junction temperatures of  $< 60^\circ\text{C}$  as shown in Figure 32 [63, 64].

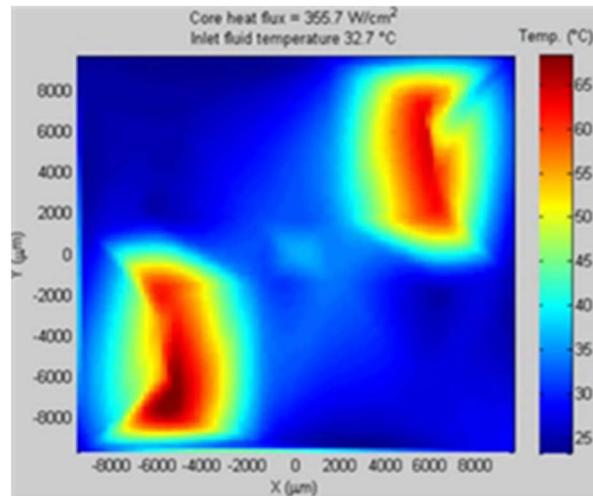


Fig 32: Interpolated visualization of the temperature profile of the thermal test vehicle [64], with two diagonal quadrants powered at  $350 \text{ W/cm}^2$ , Hot Spots  $2000 \text{ W/cm}^2$  and Background  $20 \text{ W/cm}^2$  heat flux. Total Power  $\sim 300 \text{ W}$ .

### 3.4 Advanced thermal materials for thermal management

Several new materials are being explored from a research and development standpoint for enhancement of heat spreading, thermal interfaces, and underfill materials as described in [65–67], for the various conduction layers of heterogeneously integrated (and other) packages. Examples of such materials include cubic crystals, two-dimensional layered materials, nanostructure networks and composites, molecular layers and surface functionalization, and aligned polymer structures.

Figure 33 [65] depicts a 3D chip stack using advanced materials in the conduction heat flow path. The construction in Figure 33 includes details such as the front-end-of-line (FEOL) and back-end-of-line (BEOL) device and interconnect layers as well as controlled collapse chip connections (C4) and smaller  $\mu\text{-C4}$  features. Of special note is typical epoxy-based underfill materials that fill up the gaps between the 3D stacked dies. With low intrinsic thermal conductivities of  $0.1\text{--}0.2 \text{ W/m-K}$  [65] they yield a significant thermal resistance to the heat flow as also discussed previously in Section 2.2 (3D Chip Stack with Conduction Interfaces). Thus, new underfill materials or composites need to be developed [65] with higher isotropic thermal conductivities to promote cross-plane heat conduction and die-to-die heat dissipation, while also enhancing in-plane (2D) heat spreading to mitigate hot spots in the power sources.

While Figure 33 is a valid example of a high-performance package, this opportunity for new materials is very much present for the Mobile space as well with inherent heat spreading challenges as discussed previously in Section 2.6 (Mobile Devices challenges). Indium Tin Oxide (ITO) is a material of choice for touch screens [65] in such mobile devices, but more recently materials such as carbon nanotubes (CNTs), graphene, thinSi membranes, and silver nanowires are being actively explored to solve thermal challenges [65].

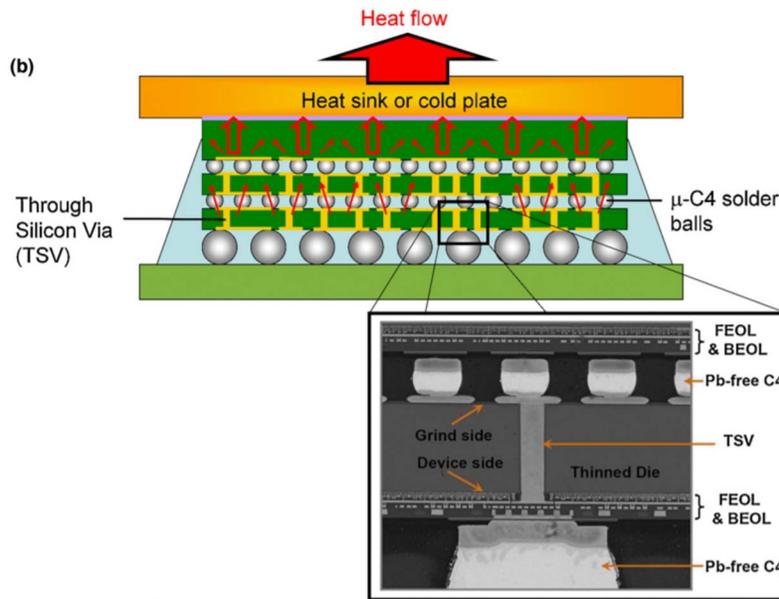


Figure 33: 3D chip stack with advanced materials in conduction heat flow path [65]

### 3.5 Thermomechanical modeling for heterogeneous integration

The power dissipation and power density in future 2D-3D packages is expected to increase and the cross-talk between different components of the package will further aggravate the thermal management challenges. This will necessitate development of a multi-physics simulation tool with closely coupled thermal, mechanical and electrical models to enable iterative simulations and robust design. This tool should allow coupling between different scales, e.g., die to package and package to system (board), to consider the effect of design at these different scales. High heat flux components of packages will require single- or two-phase liquid cooling. The models for accurate prediction of two-phase fluid flow have not been developed yet. Physics-based models in combination with machine learning tools will be needed for high fidelity prediction of performance and design of these cooling technologies.

We discuss the needs and possible approaches for next-gen modeling and simulations tools. The vast majority of thermal/thermomechanical design rules in electronics design and packaging are based on finite element simulations post-electronic design. Robust thermomechanical models are not present in the electronic design and reliability flows, thus necessitating significant margins from the designers. Here we suggest a paradigm shift to better model, optimize and design for die and package level thermomechanical effects. The primary aim of this framework is to use a repository of finite element simulations packaged through a neural network engine and abstracted into usable design models. The following workflow (Figure 34) is proposed to enable this early integration of thermal and mechanical models into design tools:

- Definition of the design space and execution of FEM simulations with combinatorial and probabilistic input parameters spanning geometrical descriptions, materials properties and interface/boundary conditions across domains.
- Training Data: Output FEM state distributions and fields (electric field, power density, temperature, stress, strain etc.). Training and validation using an artificial neural network with feed-forward deep autoencoders (DAE).
- Deployment of the validated DAEs generated in (2) to accurately predict the non-linear and statistical behavior of a design with minimum computational and setup overhead.

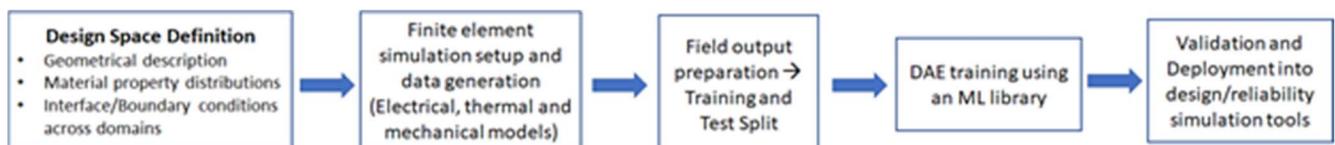


Figure 34: Thermal/Multiphysics Modeling for Heterogeneous Packages

Glass/Si based interposers or 3D packages with stacked die will allow integration of different functionalities with a wide range of power dissipation in both space and time. The efficient thermal management of hot-spots will require development of novel phase change materials, along with high-conductivity materials with anisotropic thermal

properties. In addition, next-generation packages will need novel dielectrics, insulators and conducting materials. The accurate estimation of thermal properties of these materials and their interfaces will be necessary to develop predictive models. First-principle simulations in combination with machine learning tools will be beneficial in predicting properties of these materials with specification of uncertainty, which can be input to multi-physics modeling tools to understand their effect on packages, e.g., Design Space Definition shown in the workflow above.

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